



WS 12 Aerodynamic Performance University Research Projects Wheel/Wheelarch Drag

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Drag Breakdown

In crude terms the aerodynamic drag contributions of a typical passenger car can be split 50:50 into styled and 'unstyled' surfaces.

Styled Surfaces		'Unstyled' components		
Front end	7%			
Skin friction	10%	Cooling drag/ Engine ba	y 6%	
Mirrors	3%	Underfloor	12%	
Rear end	30%	Wheels/wheelarches	32%	

Interference

These numbers are indicative only, they will vary depending on the type of vehicle and one surface interferes with another.



















Wheel Drag Component

Pfadenhauer et al. $\Delta C_D = -0.102$ Audi A3 standard floor $C_D = 0.32$ 32% Mercker et al. $\Delta C_D = -0.075$ Opel Calibra smooth floor $C_D = 0.24$ 32% C_D difference c_D 80% of drag penalty from configurations without wheels conf. with wheel arches covered (without wheels and wheel arches $\Delta c_{\rm D}$ $\Delta c_{\rm D}$ (only wheel arches covered) covered) exposed lower wheel. Ref. 0.316 0.000 0.316 Ref æ Main drag reduction from front wheel/wheelarch. 0.042 0.274 0.303 0.013 0.029 0.057 0.259 0.051 0.310 0.006 ∄(♠ 0.102 0.214 0.080 0.294 0.022 Values have been measured with stationary wheels in a different wind tunnel and have been shifted to same level as left column for ease of co

















Wheel/wheelarch Drag Measurement

- To identify aerodynamic benefits from the wheel and wheelarch ideally requires the rotation of the wheel to be simulated.
- While most European automotive wind tunnels employ moving ground with wheel rotation, no full scale facility exists in the UK.
- These facilities are expensive and the project only allocated restricted access as a sign off measure.

Many model scale wind tunnels in the UK have moving belt capability and it was decided to build a rig which would allow drag reduction opportunities from wheels and wheelarches to be investigated at a reasonable scale.

Cranfield had previously acquired the old Rover wheelrig, which had been used in the MIRA MWT (which no longer exists).



















Full Scale moving ground wind tunnel



Classic 5 belt system.

In Europe: Audi, BMW, Volvo, FKFS, Pininfarina, S2A soon Mercedes-Benz, Porsche.

DNW and BMW have wide belt systems

No UK facility















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Cranfield University Wheel rig.

Rebuilt and enlarged Rover wheel-rig

Cranfield 8' x 6' wind tunnel with moving belt ground simulation.

To assess effects of wheelarch geometry and aerodynamic aids on drag reduction of wheel and wheelarch assembly.

Drag measurement from wheel and shroud obtained separately.

Wheel drag includes rotational wheel drag, (windage loss).















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Cranfield University Wheel rig.

Wheel shroud mounted to overhead strut with internal balance.

Wheel mounted separately from outrigger struts pivoted from outside the belt. Wheel balance at outboard end of strut.





Tare drag obtained with windoff + low belt speed.

















Wheel-rig Configurations



Effect of adding wheel to basic faired body



Effect of progressively closing underside from front and rear.



Effect of progressively closing wheelarch opening from front, rear and top.



















Wheel-rig Configurations (cont)



Effect of adding full width bib spoiler - variable depth



Effect of adding wheel width fairing at front or rear of wheel- variable depth



Effect of adding wheel width fairing at front and rear of wheel

Additional investigations of simple wheel spoilers and effect of cavity volume

















Wheel-rig drag reductions

	Body	Wheel	Total
Configuration	C _{DB}	C_{DW}	C_{DT}
Baseline	0.234	0.180	0.414
	ΔC_{DB}	ΔC_{DW}	ΔC_{DT}
1.Sealed underside	-0.036	-0.009	-0.046
2.Sealed wheelarch opening	-0.003	-0.001	-0.004
3. 2+Sealed underside	-0.034	-0.014	-0.048
4.Front spoiler - 120 deep	+0.117	-0.126	-0.009
5.Front wheel fairing -120 deep	+0.008	-0.066	-0.058
6.Rear wheel fairing – 60 deep	+0.008	-0.021	-0.013
7.Front and rear wheel fairing 120 deep	+0.000	-0.071	-0.071













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Wheel-rig drag reductions

	Body	Wheel	Total
Configuration	C _{DB}	C_{DW}	C_{DT}
Baseline	0.234	0.180	0.414
	ΔC_{DB}	ΔC_{DW}	ΔC_{DT}
8.Front wheel fairing 120 deep			
+ rear fairing 60 deep	+0.008	-0.094	-0.086
9. As 8 +Sealed underside	-0.023	-0.076	-0.099
10. Rear of wheelarch cavity filled	-0.006	+0.003	-0.003
11. Wheelarch cavity fully filled	-0.015	+0.008	-0.007
Best tested configuration: Front wheel fairing 120 deep + rear fairing 60 deep + sealed underside			
+ rear of wheelarch cavity filled	+0.002	-0.106	-0.104















Example data: Effect of rear wheel fairing depth

Optimum fairing depth at approx 60mm.

Trends are very similar for both ground simulations.

Total drag higher for rotating wheel. Compare with MGWT data where the opposite occurs.



JAGUAR

open symbols - wheel stationary solid symbols – wheel rotating

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Comments

1.Simulation issues:

The simulation does not account for the crossflow experienced on front wheels of a typical car, except for the offset of the wheel in the wheelarch.

The wheel is represented by a simple chamfered thick disc. A real wheel will create more pumping action within the wheelarch.

The wheel arch is empty, except for a plain axle, whereas a real wheelarch includes suspension components, wheel hubs, brake assemblies and steering arms.

2. The large benefits will require active devices for implementation on real cars.



















Comments 2.

3. How to relate the drag reductions to real car values.

- Some uncertainty exists!
- Current approach:
- Incremental drag forces are relevant to a single front wheel.
- Forces are scaled up to account for model wheel size (~ 80%)
- Rear wheel drag is typically approximately half that of the front wheel.
- Presented coefficients based on wheel rig frontal area (0.175 m²).

Scale factor: Rig data/FS car = $1.25^2 \times 3 \times 0.175/2.25 = 0.36$.













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Conclusions

Significant aerodynamic drag reduction has been obtained from the wheel/wheelarch area of a passenger car as simulated using a simple wheel rig.

Large benefits were found with a fairing in front of the wheel and by closing the underside of the wheelarch cavity.

On real vehicles these benefits can only be realised with active devices as they intrude into ground clearance or interfere with wheel articulation.

Filling the wheelarch cavity increased drag which was not expected.

The beneficial effects of these devices still requires assessment in the real world.

















Thank You to Cranfield University for the research input to LCVTP













